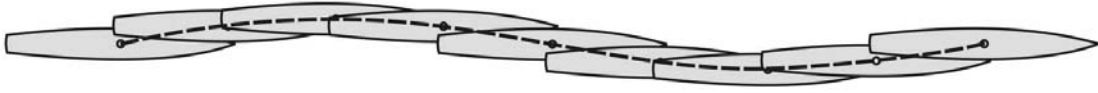


Test case 3a.6: Pure sway



Conditions

- Captive pure sway motion in still water
- Fixed (heave=5.855mm & pitch=-0.136°)
- Without rudders
- With bilge keels

Pre scribed PMM motions:

- Sway motion: $\eta_{PMM} = -2S_{mm} \sin\left(\frac{2\pi N}{60}t\right)$
- Sway velocity: $v_{PMM} = -2\left(\frac{2\pi N}{60}\right)S_{mm} \cos\left(\frac{2\pi N}{60}t\right)$
- Sway acceleration: $\dot{v}_{PMM} = 2\left(\frac{2\pi N}{60}\right)^2 S_{mm} \sin\left(\frac{2\pi N}{60}t\right)$
- Heading angle: $\psi = 0$
- Yaw rate: $r = 0$
- Yaw acceleration: $\dot{r} = 0$

| F_n [-] | R_n [-] | U_C [m/s] | N [rpm] | S_{mm} [m] | β_{max}^* [deg] | v'_{max} [-] |
|-----------|---------------------|--------------|---------------|---------------|-----------------------|----------------|
| 0.280 | 4.643×10^6 | 1.531 | 8.0210 | 0.1584 | 10.0 | 0.174 |

β_{max}^* : corresponding maximum drift angle

Items and Remarks

| Figure Number | Items | Remarks |
|---------------|---|--|
| Fig. 3a.6-1 | Axial velocity contours and cross flow vectors ($x/L_{PP}=0.135$) | 4 PMM phases; 0°, 45°, 90°, 135° To be compared with experiment results pdf |
| Fig. 3a.6-2 | Transverse velocity contours ($x/L_{PP}=0.135$) | |
| Fig. 3a.6-3 | Vertical velocity contours ($x/L_{PP}=0.135$) | |
| Fig. 3a.6-4 | Turbulent kinetic energy contours ($x/L_{PP}=0.135$) | |
| Fig. 3a.6-5 | Axial vorticity contours ($x/L_{PP}=0.135$) | |
| Fig. 3a.6-6 | Axial velocity contours and cross flow vectors ($x/L_{PP}=0.935$) | 4 PMM phases; 0°, 45°, 90°, 135° To be compared with experiment results pdf |
| Fig. 3a.6-7 | Transverse velocity contours ($x/L_{PP}=0.935$) | |
| Fig. 3a.6-8 | Vertical velocity contours ($x/L_{PP}=0.935$) | |
| Fig. 3a.6-9 | Turbulent kinetic energy contours ($x/L_{PP}=0.935$) | |
| Fig. 3a.6-10 | Axial vorticity contours ($x/L_{PP}=0.935$) | |

- Coordinate system for comparison is ship-fixed at FP on the undisturbed waterplane (x positive downstream, y positive starboard side, z positive upward).

- $$F_n = \frac{U_c}{\sqrt{gL_{PP}}}, R_n = \frac{U_c \cdot L_{PP}}{\nu}$$

where, U_c is towing carriage speed, g is the gravitational acceleration and ν is the kinematic viscosity of water.

- All quantities are non-dimensionalized with water density (ρ), carriage speed (U_c), and the length between perpendiculars (L_{PP}).

$$U = \frac{\bar{U}}{U_c}, V = \frac{\bar{V}}{U_c}, W = \frac{\bar{W}}{U_c}, k = \frac{1}{2}(\overline{uu} + \overline{vv} + \overline{ww})$$

$$\text{where, } \overline{uu} = \frac{\overline{uu}}{U_c^2}, \overline{vv} = \frac{\overline{vv}}{U_c^2}, \overline{ww} = \frac{\overline{ww}}{U_c^2}$$